

1.(15 pts) Laplace Transform OW 9.7

A system with input $x(t)$ and output $y(t)$ is described by the LDE:

$$\frac{d^2y(t)}{dt^2} + 2\frac{dy(t)}{dt} + -3y(t) = x(t) \tag{1}$$

Use the Laplace transform to solve for $y(t)$ with input $x(t) = e^{-t}u(t)$, and initial conditions $y(0^-) = 1$ and $\dot{y}(0^-) = 2$.

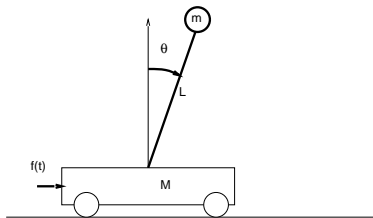


Fig. 1. Broom balancer

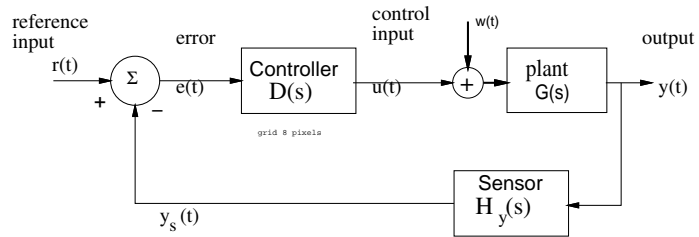


Fig. 2. Control system block diagram

2. (20 pts) (Lec 17, OW 9.5, 9.7)

The differential equation for the broom balancing system (Fig. 1) is given by

$$\frac{1}{3}(4M + m)L\ddot{\theta}(t) = (m + M)g\theta(t) - f(t)$$

under the assumption that $|\theta| \ll 1$ and the approximation $\sin \theta = \theta$, and $f(t)$ is the force applied to move the cart.

- Find the Laplace transform relation for the broom balancing system including both the zero state response and the zero input response.
- Suppose you can measure $\theta(t)$. You want to balance the broom by choosing $f(t)$ in feedback form as $f(t) = \alpha L\theta(t)$. Will this scheme result in balancing the broom, i.e. so that $\theta(t) \rightarrow 0$ as $t \rightarrow \infty$, for any small initial condition $\theta(0^-)$ and $\dot{\theta}(0^-)$? Explain why or why not.
- Suppose you can measure $\theta(t)$ and $\dot{\theta}(t)$. You want to balance the broom by choosing f in feedback form as $f(t) = \alpha L\theta(t) + \beta L\dot{\theta}(t)$. For what values of α and β will this scheme result in balancing the broom, i.e. so that $\theta(t) \rightarrow 0$ as $t \rightarrow \infty$, for any small initial condition $\theta(0^-)$ and $\dot{\theta}(0^-)$.

3. (15 pts) Feedback Control (Lec 17,18, OW 11-11.2)

Consider the feedback system of Fig.2, with $w(t) = 0$, $D(s) = 1$. Determine the closed loop transfer function $\frac{Y(s)}{R(s)}$, and closed-loop impulse response (i.e. let $r(t) = \delta(t)$) for each of the following system functions in forward and feedback paths:

- $G(s) = \frac{1}{(s+1)(s+4)}$, $H_y(s) = 1$.
- $G(s) = \frac{1}{(s+4)}$, $H_y(s) = \frac{1}{s+1}$.
- $G(s) = \frac{1}{2}$, $H_y(s) = e^{-s/3}$.

4. (25 pts) Feedback Controller Design (Lec 17,18, OW 11-11.2)

Consider the feedback system of Fig.2, with $w(t) = 0$, $H_y(s) = 1$.

- a. Suppose $G(s) = \frac{\alpha}{s+\alpha}$ with $\alpha \neq 0$. Show that with proportional control, $D(s) = K$, K can be chosen to stabilize the system, and that $e(t)$ will not tend to zero with $r(t) = u(t)$.
- b. Suppose $G(s) = \frac{\alpha}{s+\alpha}$ with $\alpha \neq 0$. Show that with proportional-plus-integral (PI) control, $D(s) = K_1 + \frac{K_2}{s}$, then K_1, K_2 can be chosen to stabilize the system, and that $e(t)$ will tend to zero with $r(t) = u(t)$.
- c. Suppose $G(s) = \frac{1}{(s-1)^2}$. Show that with proportional-plus-integral-plus-derivative control, $D(s) = K_1 + \frac{K_2}{s} + K_3s$, then K_1, K_2, K_3 can be chosen to stabilize the system, and that $e(t)$ will tend to zero with $r(t) = u(t)$. Also show that the system can not be stabilized with a PI controller.

5. (25 pts) Gain and Phase Margin (Lec 18,19, OW 6.5, 9.4, 11.5)

Consider the feedback system of Fig.2, with $w(t) = 0$, $D(s) = 1$. For each part below:

- a) Determine the closed loop transfer function $\frac{Y(s)}{R(s)}$.
- b) sketch the pole-zero diagram for $G(s)H_y(s)$.
- c) Sketch the magnitude and phase Bode plots (e.g. Fig. 11.27) of $G(j\omega)H_y(j\omega)$
- d) Roughly estimate the gain and phase margin.

i) $G(s) = \frac{10s+1}{s^2+s+1}$, $H_y(s) = 1$.

ii) $G(s) = \frac{s/10+1}{s^2+s+1}$, $H_y(s) = 1$.

iii) $G(s) = \frac{1}{(s+3)^3}$, $H_y(s) = \frac{1}{s+3}$.

iv) $G(s) = \frac{1}{(s+1)^2(s+10)}$, $H_y(s) = 100$.