

Get Potential Energy & Frequency

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Folded-beam suspension $60\mu\text{m}$ $Q=100k$

Shuttle w/ mass M_s
Area: $4,000\mu\text{m}^2$

Folding truss w/ mass $M_t/2$

Anchor $h = \text{thickness} = 2\mu\text{m}$

$K_{eq} = M_{eq} \cdot C_{eq} = \infty$

$K_{eq}(\text{truss}) = 19.2 \text{ N/m}$
 $M_{eq}(\text{truss}) = 8.64 \times 10^{-11} \text{ kg}$
 $C_{eq}(\text{truss}) = 4.08 \times 10^{-10} \text{ kg/s}$

$K_{eq}(\text{shuttle}) = 4.8 \text{ N/m}$
 $M_{eq}(\text{shuttle}) = 2.16 \times 10^{-11} \text{ kg}$
 $C_{eq}(\text{shuttle}) = 1.02 \times 10^{-10} \text{ kg/s}$

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3CC $3\lambda/4$ Bridged μ Mechanical Filter

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Performance:

- $f_o = 9\text{MHz}$, $BW = 20\text{kHz}$, $PBW = 0.2\%$
- I.L. = 2.79dB, Stop. Rej. = 51dB
- 20dB S.F. = 1.95, 40dB S.F. = 6.45

Transmission [dB]

Frequency [MHz]

Design:

- $L_f = 40\mu\text{m}$
- $W_f = 6.5\mu\text{m}$
- $h_f = 2\mu\text{m}$
- $L_c = 3.5\mu\text{m}$
- $L_b = 1.6\mu\text{m}$
- $V_p = 10.47\text{V}$
- $P = -5\text{dBm}$
- $R_{qi} = R_{qo} = 12\text{k}\Omega$

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Micromechanical Filter Circuit

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Input R_Q

Bridging Beam $3\lambda/4$

Coupling Beam $\lambda/4$

Resonator $\lambda/4$

Output R_Q

v_i , V_P , v_o , ω

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Micromechanical Filter Circuit

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Input R_Q

Bridging Beam $3\lambda/4$

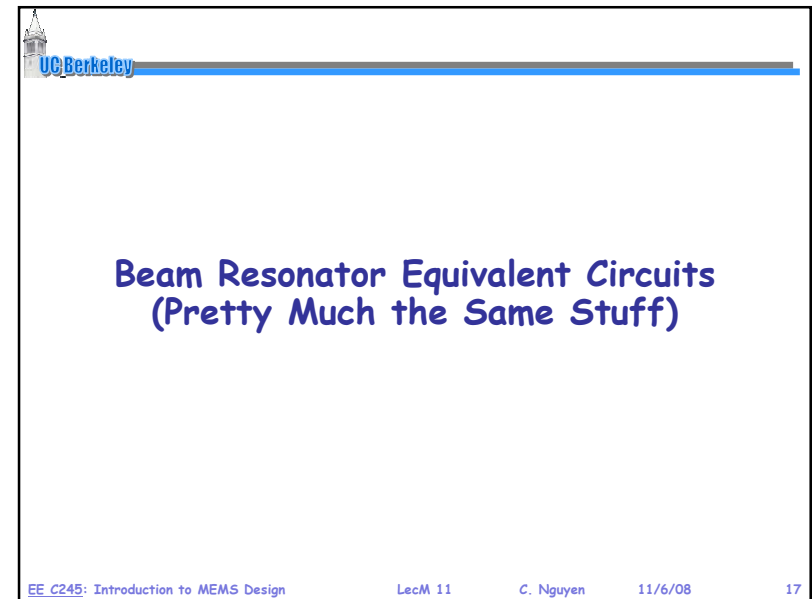
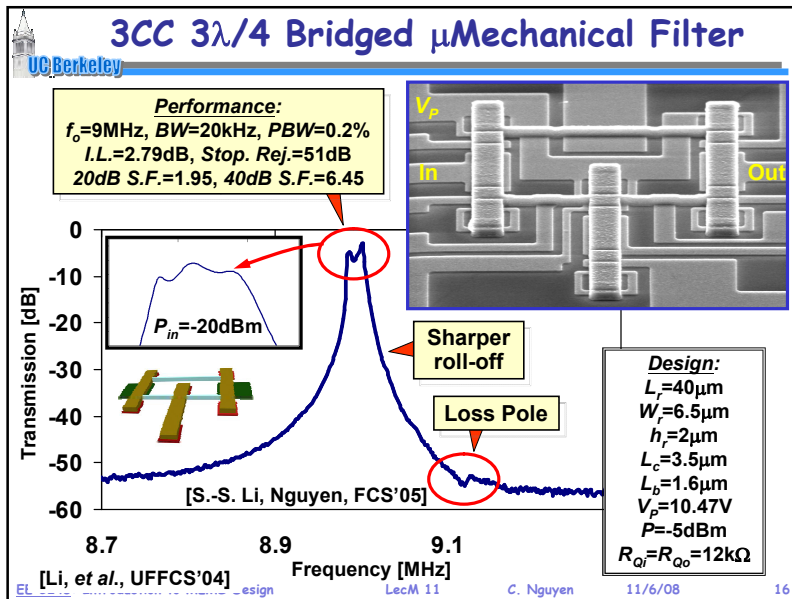
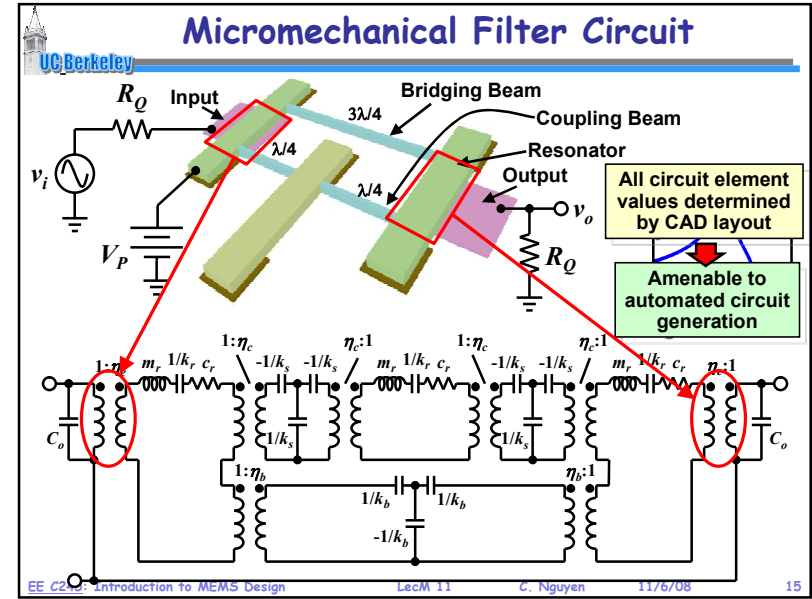
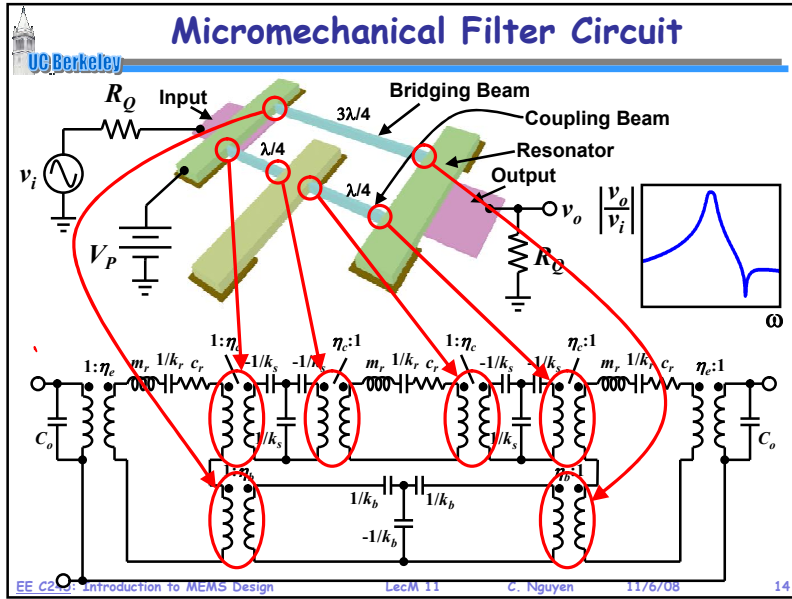
Coupling Beam $\lambda/4$

Resonator $\lambda/4$

Output R_Q

v_i , V_P , v_o , ω

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Equivalent Dynamic Mass

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- Once the mode shape is known, the lumped parameter equivalent circuit can then be specified
- Determine the equivalent mass at a specific location x using knowledge of kinetic energy and velocity

Maximum Kinetic Energy \rightarrow $\frac{1}{2} \rho A \int_0^l v^2(x) dx$

Density \rightarrow $\frac{1}{2} \rho A \int_0^l v^2(x) dx$

Equivalent Mass = $M_{eq\ x} = \frac{K.E.}{\frac{1}{2} v_x^2} = \frac{\frac{1}{2} \rho A \int_0^l v^2(x) dx}{\frac{1}{2} v_x^2}$

Maximum Velocity @ location x \rightarrow $\frac{1}{2} v_x^2$

Maximum Velocity Function \rightarrow $\frac{1}{2} v_x^2$

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Equivalent Dynamic Mass

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- We know the mode shape, so we can write expressions for displacement and velocity at resonance

Displacement: $u(x) = B [S(\cosh kx + \cos kx) + (\sinh kx + \sin kx)]$, $S = \frac{A}{B}$

$[v(x) = \omega u(x)] \Rightarrow M_{eq}(x) = \frac{KE_{max}}{\frac{1}{2} [v(x)]^2} = \frac{\frac{1}{2} \rho A \int_0^l \omega^2 [u(x')]^2 dx'}{\frac{1}{2} \omega^2 [u(x)]^2}$

$$M_{eq}(x) = \frac{\rho A \int_0^l B^2 [S(\cosh kx' + \cos kx') + (\sinh kx' + \sin kx')]^2 dx'}{B^2 [S(\cosh kx + \cos kx) + (\sinh kx + \sin kx)]^2}$$

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Equivalent Dynamic Stiffness & Damping

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- Stiffness then follows directly from knowledge of mass and resonance frequency

$$\omega_0 = \sqrt{\frac{K_{eq}(x)}{M_{eq}(x)}} \rightarrow K_{eq}(x) = \omega_0^2 M_{eq}(x)$$

- And damping also follows readily

$$Q = \frac{\omega_0 M_{eq}(x)}{C_{eq}(x)} \rightarrow C_{eq}(x) = \frac{\omega_0 M_{eq}(x)}{Q} = \frac{\sqrt{K_{eq}(x) M_{eq}(x)}}{Q}$$

↑ damping

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Equivalent Lumped Mechanical Circuit

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$K_{eq}(x) = \omega_0^2 M_{eq}(x)$

$M_{eq}(x) = \frac{\rho A \int_0^l [u(x')]^2 dx'}{[u(x)]^2}$

$C_{eq}(x) = \frac{\omega_0 M_{eq}(x)}{Q}$

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