

Lecture 20: Electrical Stiffness

- Announcements:
- Module 12 on Capacitive Transducers online
- HW#5 online and due Thursday, April 13
- In mid-class (to make sure everyone is here)
 - ↳ Project introduction today
 - ↳ Midterm exams coming back today
 - ↳ Z-scores presented as well, at the end of class

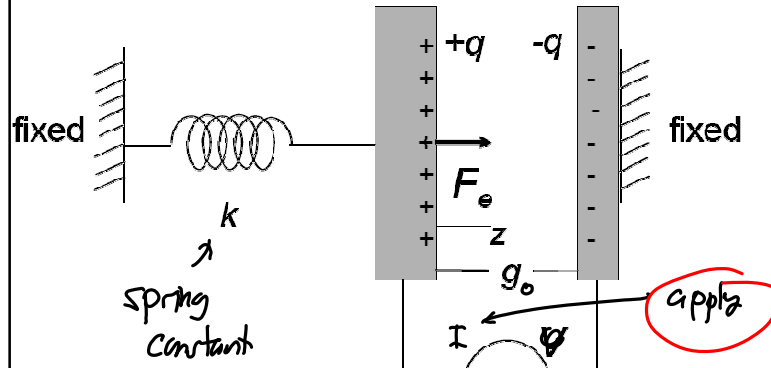
- Reading: Senturia, Chpt. 5, Chpt. 6

- Lecture Topics:
 - ↳ Energy Conserving Transducers
 - Charge Control
 - Voltage Control
 - ↳ Parallel-Plate Capacitive Transducers
 - Linearizing Capacitive Actuators
 - Electrical Stiffness
 - ↳ Electrostatic Comb-Drive
 - 1st Order Analysis
 - 2nd Order Analysis

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- Last Time:
 - Determined pull-in voltage



Charge-Control of a Spring-Suspended C



Force generated by charge q (supplied by current I):

$$F_e = \left. \frac{\partial W(q, g)}{\partial g} \right|_q = \frac{q^2}{2\epsilon A}$$

Restoring force of spring, $F_{spring} = kz = F_e$
 Equilibrium

The gap:

$$g = g_0 - z = g_0 - \frac{F_e}{k} = \left[g_0 - \frac{1}{2} \frac{q^2}{\epsilon A} \frac{1}{k} = g \right]$$

↳ $q \uparrow$ can drive $g \rightarrow 0$ in continuous fashion

$$V = \frac{q}{C} = \frac{q}{\epsilon A/g} = \frac{q}{\epsilon A} \left(g_0 - \frac{1}{2} \frac{q^2}{\epsilon A} \frac{1}{k} \right) = V \leftarrow V \text{ as } g \downarrow$$

Voltage-Control of a Suspended C

fixed k F_e z g_0 initial gap spacing V fixed

But now:
 $F_e = \frac{\partial W'(V, g)}{\partial g} \Big|_q \rightarrow F_e = \frac{1}{2} \frac{\epsilon A}{g^2} V^2$

And the gap:
 $g = g_0 - z = g_0 - \frac{F_e}{k} = g_0 - \frac{1}{2} \frac{\epsilon A}{g^2} \frac{V^2}{k} = g$

g shows up on both sides!

If $V \uparrow \rightarrow g \downarrow \rightarrow F_e \uparrow$
 (+) Feedback!

If loop gain > 1 , then this will go unstable!
 plates will collapse!
 (into the electrode)

Charge: (for a stable gap)
 $q = \frac{\partial W'(V, g)}{\partial V} \Big|_g = CV \sqrt{\quad}$ (as expected)

Stability Analysis

\Rightarrow determine under what conditions voltage-control will cause collapse of the plates:

$$F_{net} = F_e - F_{spring} = \frac{\epsilon AV^2}{2g^2} - \underbrace{k(g_0 - g)}_{spring}$$

What happens when I change g by a small increment dg ?

\hookrightarrow get an increment in the net attractive force F_{net}

$$\frac{dF_{net}}{dg} = \frac{\partial F_{net}}{\partial g} dg = \left[-\frac{\epsilon AV^2}{g^3} + k \right] dg$$

If $g \downarrow + dg = (-)$, then for stability need $F_{net} \downarrow \rightarrow dF_{net} = (-)$

This must be (+)! \rightarrow otherwise, the plates collapse!

Thus: $k > \frac{\epsilon AV^2}{g^3}$ (for a stable uncollapsed system)

Pull-in Voltage V_{PI} & Pull-in Gap g_{PI}

$V_{PI} \triangleq$ voltage @ which plates collapse
 $g_{PI} \triangleq$ gap @ " " "

The plates go unstable when:

$$k = \frac{\epsilon A V_{PI}^2}{g_{PI}^3} \quad (1)$$

$$F_{net} = 0 = \frac{\epsilon A V_{PI}^2}{2g_{PI}^2} - k(g_0 - g_{PI}) \quad (2)$$

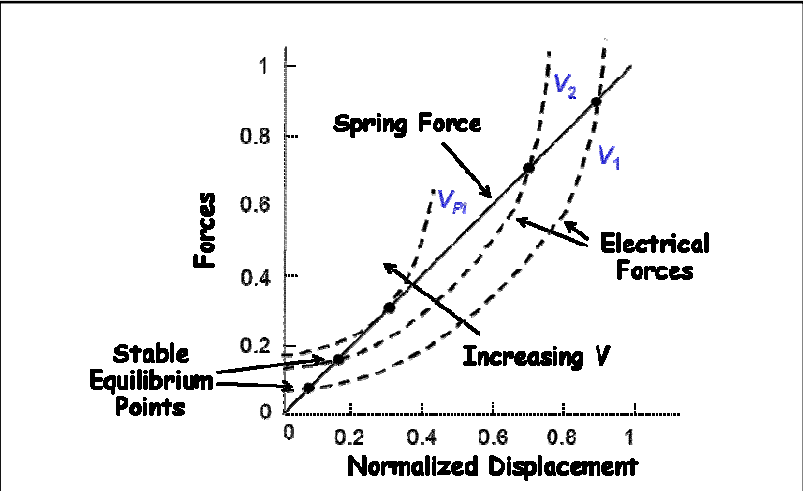
Substitute (1) into (2):

$$0 = \frac{\cancel{\epsilon A V_{PI}^2}}{2g_{PI}^2} - \frac{\cancel{\epsilon A V_{PI}^2}}{g_{PI}^3} (g_0 - g_{PI})$$

$$\frac{g_0 - g_{PI}}{g_{PI}} = \frac{1}{2} \rightarrow g_0 = \frac{3}{2} g_{PI}$$

$$\therefore \boxed{g_{PI} = \frac{2}{3} g_0}$$

When the gap is driven by a plate
to (2/3) the initial gap \rightarrow collapse!

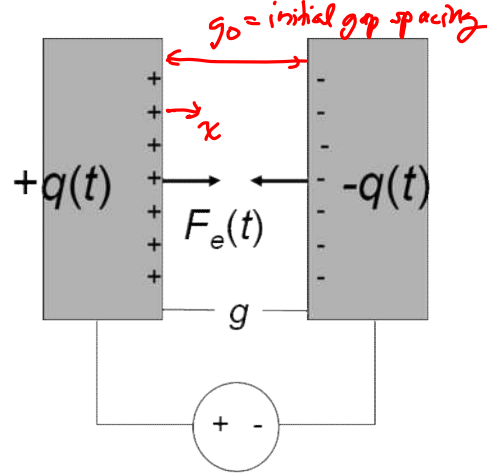
$$V_{PI} = \sqrt{\frac{k g_{PI}^3}{\epsilon A}} \rightarrow \boxed{V_{PI} = \sqrt{\frac{8}{27} \frac{k g_0^3}{\epsilon A}}}$$


- Advantages of Electrostatic Actuators:
- Easy to manufacture in micromachining processes, since conductors and air gaps are all that's needed \rightarrow low cost!
 - Energy conserving \rightarrow only parasitic energy loss through I^2R losses in conductors and interconnects
 - Variety of geometries available that allow tailoring of the relationships between voltage, force, and displacement
 - Electrostatic forces can become very large when dimensions shrink \rightarrow electrostatics scales well!
 - Same capacitive structures can be used for both drive and sense of velocity or displacement
 - Simplicity of transducer greatly reduces mechanical energy losses, allowing the highest Q's for resonant structures

Disadvantages of Electrostatic Actuators:

- Nonlinear voltage-to-force transfer function
- Relatively weak compared with other transducers (e.g., piezoelectric), but things get better as dimensions scale

Linearizing the Voltage-to-Force Transfer Fcn



$v(t) = V_P + v_i(t)$
 DC-bias signal (AC)

$$F_e(t) = \frac{\partial W'}{\partial x} = \frac{\partial}{\partial x} \left[\frac{1}{2} C [v(t)]^2 \right]$$

$$= \frac{1}{2} \frac{\partial C}{\partial x} [v(t)]^2 = \frac{1}{2} \frac{\partial C}{\partial x} [V_P + v_i(t)]^2$$

$$= \frac{1}{2} [V_P^2 + 2V_P v_i(t) + v_i(t)^2] \frac{\partial C}{\partial x}$$

$[V_P \gg v_i(t)] \Rightarrow \frac{1}{2} V_P^2 \frac{\partial C}{\partial x} + V_P \frac{\partial C}{\partial x} v_i(t)$
 DC offset AC Drive Signal

$C_0 = \frac{\epsilon A}{g_0} \rightarrow C(x) = \frac{\epsilon A}{g_0 - x} = C_0 \left(1 - \frac{x}{g_0}\right)^{-1}$ ↓ binomial theorem

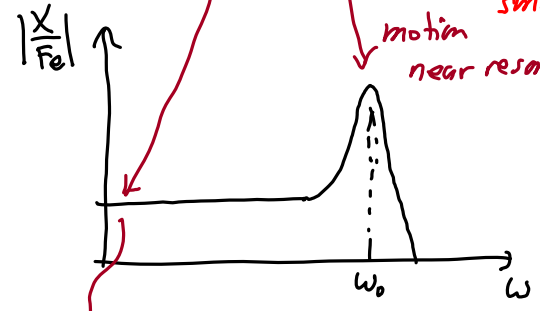
$[x \ll g_0] \Rightarrow \approx C_0 \left(1 + \frac{x}{g_0}\right)$

$\therefore \frac{\partial C}{\partial x} = \frac{C_0}{g_0} = \frac{\epsilon A}{g_0^2}$

$\Rightarrow F_e(t) = \frac{1}{2} \frac{C_0}{g_0} V_P^2 + V_P \frac{C_0}{g_0} v_i(t)$

DC offset ~ constant for small amplitudes \therefore this is a linear dependence!

$v_i \ll V_P$ can still initiate a lot of motion near resonance!



But still must worry about pull-in, V_{PI} !

Can Cancel the DC offset via Differential Symmetry

$$F_{net}(t) = F_{er}(t) - F_{el}(t)$$

$$= \frac{1}{2} \frac{\partial C}{\partial x} \{ [V_R(t)]^2 - [V_L(t)]^2 \}$$

$$= \frac{1}{2} \frac{\partial C}{\partial x} \{ \cancel{V_P^2} + 2V_P V(t) + \cancel{[V(t)]^2} - (\cancel{V_P^2} - 2V_P V(t) + \cancel{[V(t)]^2}) \}$$

$$\therefore F_{net}(t) = 2V_P \frac{\partial C}{\partial x} N(t) = 2V_P \frac{\epsilon_0}{g_0} N(t)$$

quite linear w/ $N(t)$!

still an approximation to $\frac{\partial C}{\partial x}$

Nonlinearity Still Effects Us (even though we're using small signals)

$$C_1(x) = \frac{\epsilon A}{d_1 + x} = C_0 \left(1 + \frac{x}{d_1}\right)^{-1} \rightarrow \frac{\partial C_1}{\partial x} = -\frac{C_0}{d_1} \left(1 + \frac{x}{d_1}\right)^{-2}$$

[Expand into Taylor Series]

$$\frac{\partial C_1}{\partial x} = -\frac{C_0}{d_1} \left(1 + A_1 x + A_2 x^2 + A_3 x^3 + \dots\right)$$

where $A_1 = -\frac{2}{d_1}$, $A_2 = \frac{3}{d_1^2}$, $A_3 = -\frac{4}{d_1^3}$, ...

$$F_{d1} = \frac{1}{2} \frac{\partial C_1}{\partial x} (V_p - V_1 - v_1)^2 = \frac{1}{2} \frac{\partial C}{\partial x} (V_{p1} - v_1)^2$$

$V_{p1} = V_p - V_1$

[small displacements: $x \ll d_1$]

$$F_{d1} = \frac{1}{2} \left(-\frac{C_0}{d_1} \right) (1 + A_1 x) (V_{p1}^2 + 2V_{p1}v_1 + v_1^2)$$

$v_1 = v_1 \cos^2 \omega t$

$$= \frac{1}{2} \left(-\frac{C_0}{d_0} \right) \left\{ V_{p1}^2 + 2V_{p1}v_1 + v_1^2 + A_1 V_{p1}^2 x - 2A_1 V_{p1} x v_1 + A_1 x v_1^2 \right\}$$

ω_0

Resonance: $\left| \frac{x}{F_d} \right|$