

Lecture 14: Beam Combos I

- Announcements:
- HW#4 online, due Tuesday, 3/19, 9 a.m.
- Midterm Exam about 2 weeks away, Thursday, March 21, 11-12:30 p.m., 293 Cory (right here)

 • Reading: Senturia, Chpt. 9

• Lecture Topics:

↳ Bending of beams

- ↳ Cantilever beam under small deflections
- ↳ Combining cantilevers in series and parallel
- ↳ Folded suspensions
- ↳ Design implications of residual stress and stress gradients

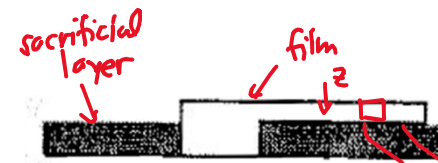
 • Last Time:

- Working through stress gradients
- Continue with this



Stress Gradient in a Cantilever

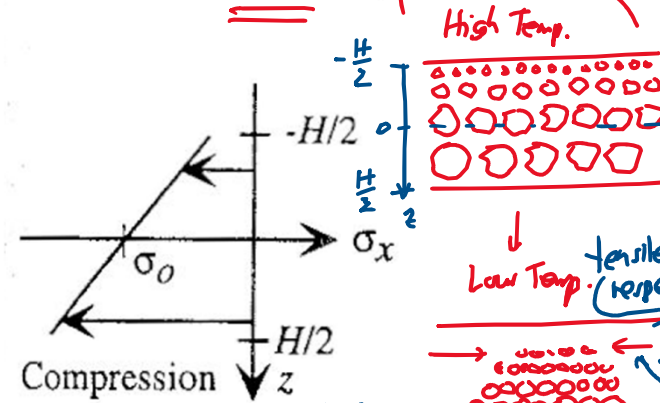
⇒ study beam bending due to a stress gradient

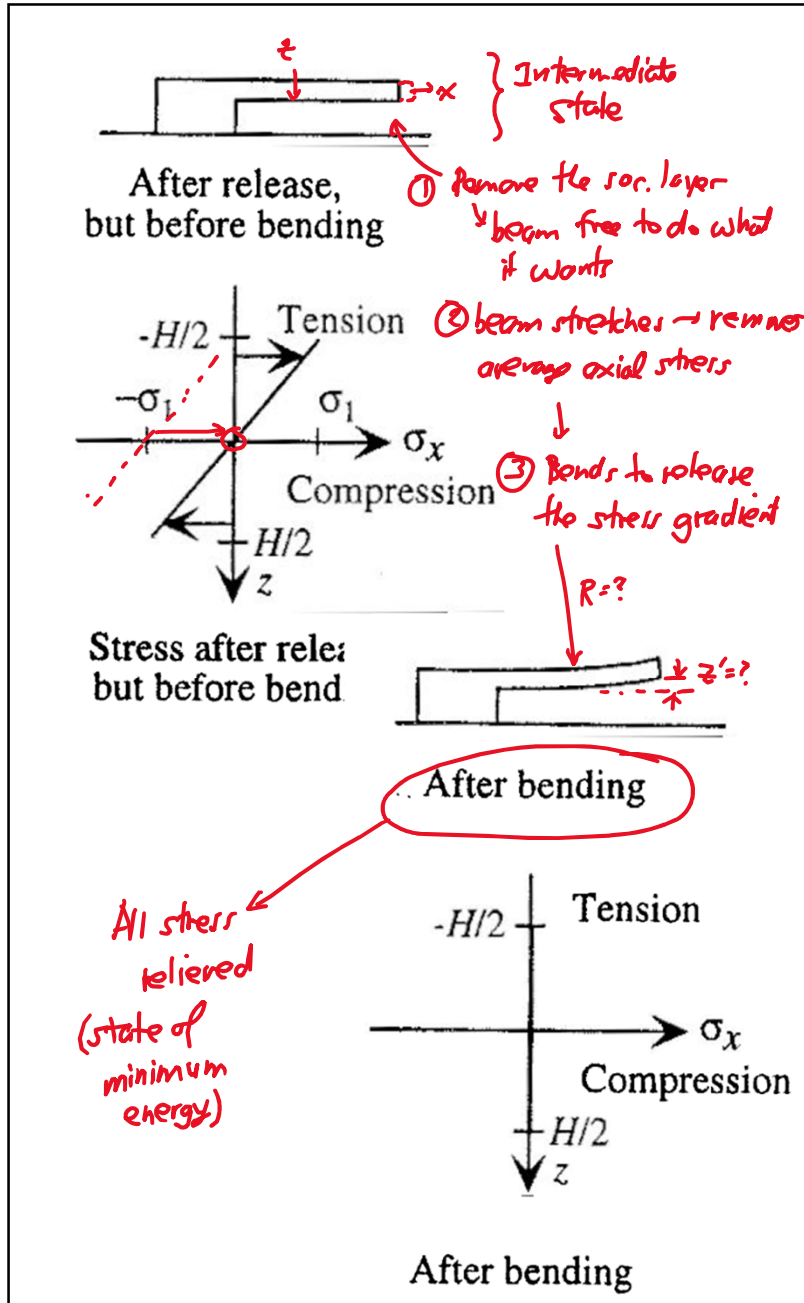


① Deposit film @ high temp.

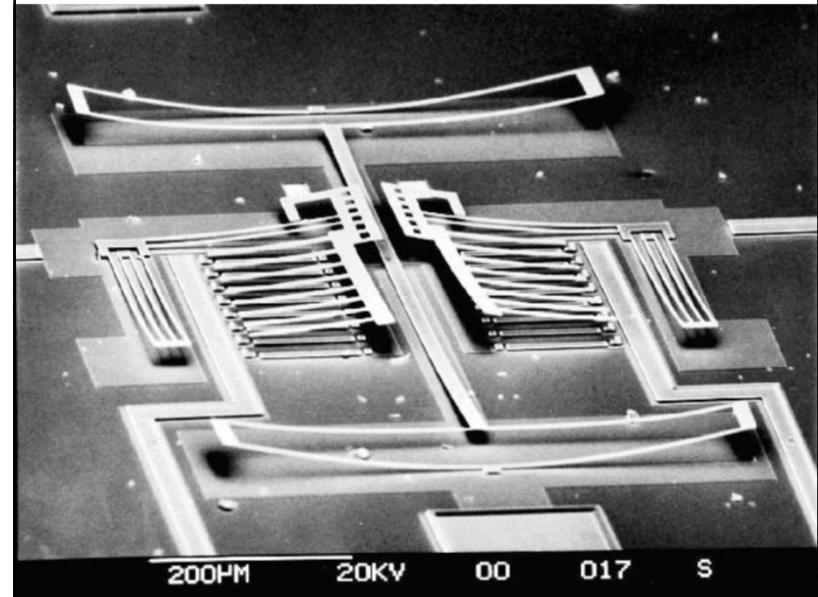
② Cool it down.

Before release





• ... and the result:



• Need to quantify this

Bending Due to Stress Gradient

Goal: Find the radius of curvature $\rightarrow z'$

Prior to release, axial stress: $\sigma = \sigma_0 - \frac{\sigma_1}{(H/2)} z$

The internal moment:

$$M_x = \int_{-\frac{H}{2}}^{\frac{H}{2}} [(Wdz) \cdot \sigma] \cdot z = \int_{-\frac{H}{2}}^{\frac{H}{2}} (z\sigma_0 - \frac{\sigma_1 z^2}{(H/2)}) dz$$

$$= W \left(\frac{1}{2} \sigma_0 z^2 - \frac{2\sigma_1 z^3}{3H} \right) \Big|_{-\frac{H}{2}}^{\frac{H}{2}}$$

$$= W \left(\frac{1}{3} \sigma_0 \frac{H^2}{4} - \frac{2}{3} \sigma_1 \frac{H^2}{8} - \frac{1}{2} \sigma_0 \frac{H^2}{4} + \frac{2}{3} \sigma_1 \frac{H^2}{8} \right)$$

average stress cancels out

$$M_x = -\frac{1}{6} \sigma_1 W H^2$$

Thus, the radius of curvature:

$$\frac{1}{R} = -\frac{M_x}{E'I} \rightarrow R = \frac{E'I}{M_x} = \frac{1}{2} \frac{E'H}{\sigma_1}$$

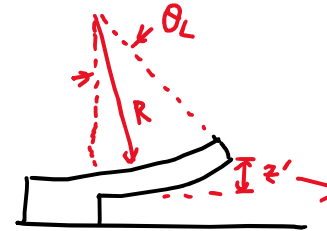
Biaxial Modulus

$$[I = \frac{1}{12} W H^3]$$

$$R = \frac{1}{2} \frac{E}{1-\nu} \frac{H}{\sigma_1}$$

Radius of Curvature for a Cantilever w/ a Stress Gradient

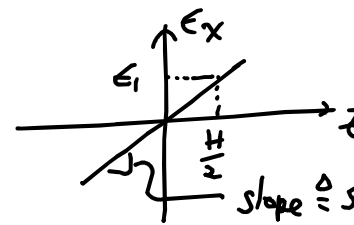
Radius of Curvature $\rightarrow z'$



integrate over θ_L to get z'

Do in homework...

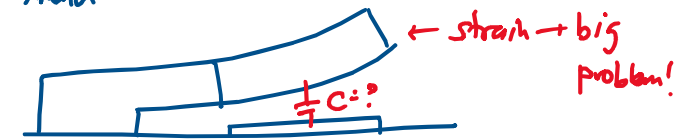
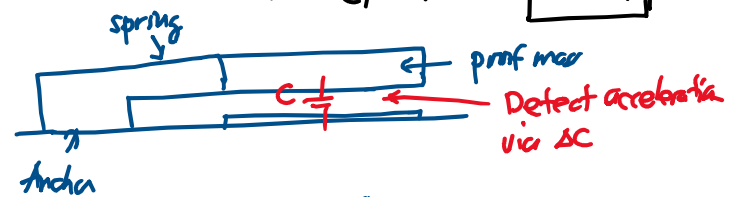
Definition: Strain Gradient



slope $\hat{=}$ strain gradient = Γ

$$\Gamma = \frac{\epsilon_1}{(H/2)}$$

$$R = \frac{1}{2} \frac{E}{1-\nu} \frac{H}{\sigma_1} = \frac{H}{2} \frac{E'}{\sigma_1} = \frac{(H/2)}{\epsilon_1} = \frac{1}{\Gamma} \rightarrow \boxed{\Gamma = \frac{1}{R}} \checkmark$$



Folded Beam Suspension

⇒ Module 8, slide 22 :

① Deposit @ high T.
stress free

② Cool to RT + stress!

Folded Beams eliminate these buckling forces.
to 1st order!

How to Defend Against the Above:

- ① Δ process parameters for deposition of structural material
↓ never perfect
- ② Solution: folded-beams!

Analyzing an Interconnected Ensemble of Beams (Springs) & Masses

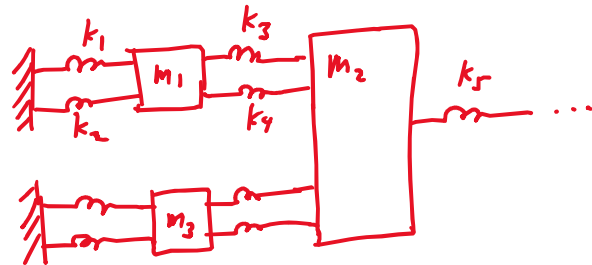
Typical Questions: → all demand knowledge of $x = f(F)$

- ① How does the structure move in response to a force at a specific location. → in turn requires we know stiffness!
- ② What is the frequency response to an AC force that applied at a specific location.
- ③ Noise?
- ④ Response to environmental stimuli? (e.g., rotation)
- ⑤ How does stress affect the behavior of the structure?

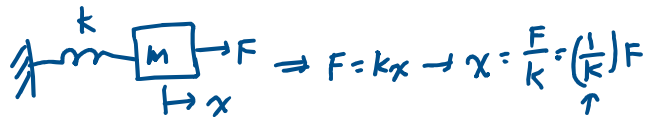
Procedure:

- Build the ckt. (extract the ckt.)
↳ in the x-direction (for this example)

Anchra

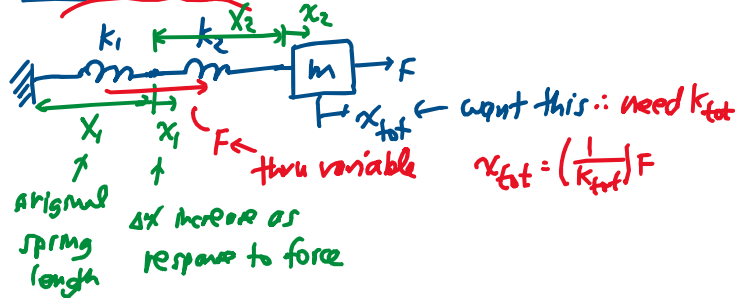


- Analyze to get $x \cdot f(F)$ force
displacement



series because one must go thru both k_1 & k_2 to get from anchor to forcing pt.

- Case 1: series connection of springs forcing pt.



$$x_{tot} = x_1 + x_2 = F \left(\frac{1}{k_1} + \frac{1}{k_2} \right) = \frac{F}{\left(\frac{1}{k_1} + \frac{1}{k_2} \right)} = \frac{F}{k_{tot}}$$

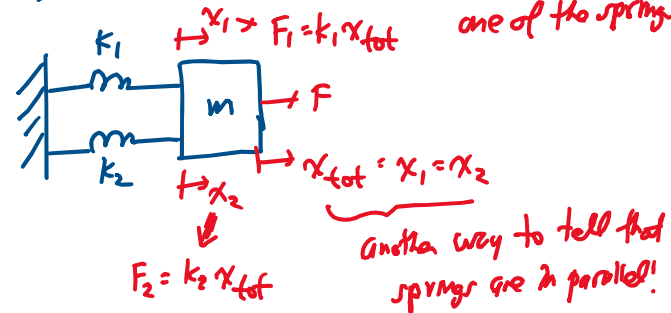
$\left[x_1 = \frac{F}{k_1}, x_2 = \frac{F}{k_2} \right]$

"||" operator $\triangleq A || B = \frac{1}{\frac{1}{A} + \frac{1}{B}} = \frac{AB}{A+B}$

$k_{tot} = k_1 || k_2$ (for k_1 & k_2 in series)

For EE's: $\frac{1}{C_1} || \frac{1}{C_2} \equiv \frac{1}{C_{tot} = C_1 C_2}$
springs combine like capacitors

- Case 2: parallel springs to forcing pt. via only one of the springs



$$F = F_1 + F_2 = (k_1 + k_2) x_{tot}$$

$k_{tot} = k_1 + k_2$ (for k_1 & k_2 in parallel)

